

THERMAL AND KINETIC PHYSICS (PHY 214)

EXERCISE 8 : WEEK 9

OUTLINE SOLUTIONS

QUESTION 1: (20 marks)

(a) We consider an infinitesimal change in temperature dT at constant total magnetization, \mathcal{M} . The thermodynamic identity tells us that for any change

$$dU = TdS + B_0d\mathcal{M}$$

so for a change at constant \mathcal{M} we have $dU = TdS$ and if we divide this equation by dT at constant \mathcal{M} we get

$$\left. \frac{dU}{dT} \right|_{\mathcal{M}} = T \left. \frac{dS}{dT} \right|_{\mathcal{M}}$$

Identifying the partial derivatives gives

$$\left(\frac{\partial U}{\partial T} \right)_{\mathcal{M}} = T \left(\frac{\partial S}{\partial T} \right)_{\mathcal{M}} = C_{\mathcal{M}} \quad \text{[3mks]}$$

Alternatively consider a change $d\mathcal{M}$ at constant T so we may divide the thermodynamic identity by $d\mathcal{M}$, remembering that all changes are at constant T , to get

$$\left. \frac{dU}{d\mathcal{M}} \right|_T = T \left. \frac{dS}{d\mathcal{M}} \right|_T + B_0$$

However, if we regard U as a function of \mathcal{M} , then we see that

$$\left. \frac{dU}{d\mathcal{M}} \right|_T = \left(\frac{\partial U}{\partial \mathcal{M}} \right)_T$$

giving the required result

$$\left(\frac{\partial U}{\partial \mathcal{M}} \right)_T = T \left(\frac{\partial S}{\partial \mathcal{M}} \right)_T + B_0 \quad \text{[2mks]}$$

(b)

i) Using the definition, $F = U - TS$ and considering a small change in the Helmholtz free energy we may write

$$dF = dU - TdS - SdT = TdS + B_0d\mathcal{M} - TdS - SdT$$

$$dF = -SdT + B_0d\mathcal{M} \quad [3\text{mks}]$$

ii) The above indicates that F has as its natural variables T and \mathcal{M} , so that

$$dF = \left(\frac{\partial F}{\partial T}\right)_{\mathcal{M}} dT + \left(\frac{\partial F}{\partial \mathcal{M}}\right)_T d\mathcal{M} \quad [2\text{mks}]$$

so that, identifying the two expressions for dF we have

$$-S = \left(\frac{\partial F}{\partial T}\right)_{\mathcal{M}} \quad B_0 = \left(\frac{\partial F}{\partial \mathcal{M}}\right)_T \quad [2\text{mks}]$$

Since the mixed second partial derivatives of a perfect differential obey

$$\left(\frac{\partial^2 F}{\partial \mathcal{M} \partial T}\right) = \left(\frac{\partial^2 F}{\partial T \partial \mathcal{M}}\right)$$

we deduce

$$\left(\frac{\partial^2 F}{\partial \mathcal{M} \partial T}\right) = \frac{\partial}{\partial \mathcal{M}} \frac{\partial F}{\partial T} = -\left(\frac{\partial S}{\partial \mathcal{M}}\right)_T = \frac{\partial}{\partial T} \frac{\partial F}{\partial \mathcal{M}} = \left(\frac{\partial B_0}{\partial T}\right)_{\mathcal{M}} \quad [2\text{mks}]$$

$$-\left(\frac{\partial S}{\partial \mathcal{M}}\right)_T = \left(\frac{\partial B_0}{\partial T}\right)_{\mathcal{M}}$$

and substituting the above gives the required result

$$\left(\frac{\partial U}{\partial \mathcal{M}}\right)_T = -T \left(\frac{\partial B_0}{\partial T}\right)_{\mathcal{M}} + B_0 \quad [1\text{mk}]$$

(c) Using the equation of state we can evaluate the derivative of B_0 with respect to T ,

$$B_0 = \frac{\mu_0 T}{CV} \mathcal{M} \rightarrow \left(\frac{\partial B_0}{\partial T}\right)_{\mathcal{M}} = \frac{\mu_0}{CV} \mathcal{M} \quad [2\text{mks}]$$

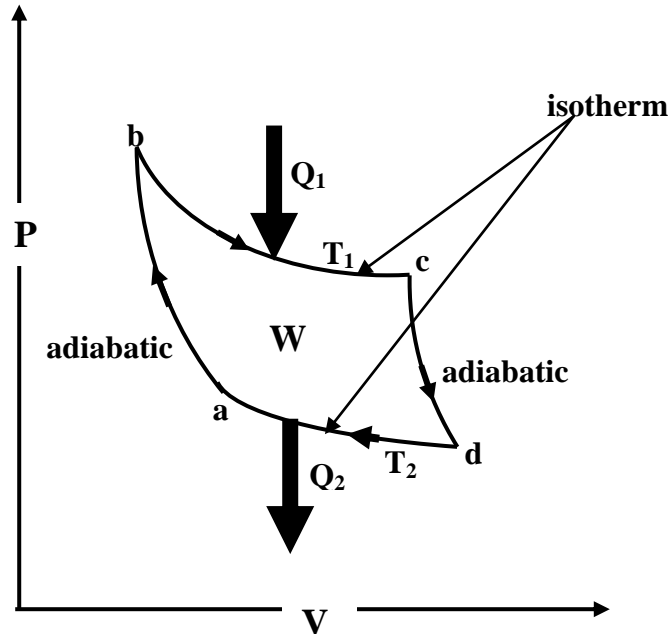
and substituting into ii) above gives

$$\left(\frac{\partial U}{\partial \mathcal{M}}\right)_T = -T \left(\frac{\mu_0}{CV} \mathcal{M}\right) + B_0 = -B_0 + B_0 = 0 \quad [2\text{mks}]$$

Since this derivative is identically zero, U cannot depend on \mathcal{M} and must be a function of T alone. [1mk]

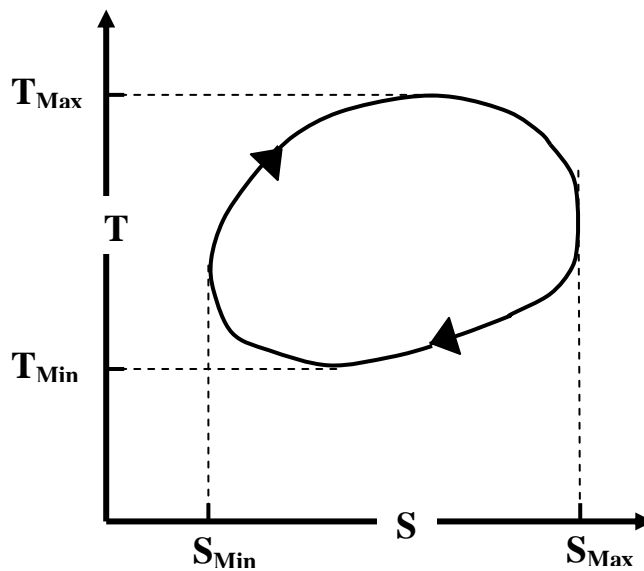
QUESTION 2: (20 marks)

(a) The Carnot cycle consists of two isothermal paths on which heat is exchanged and two adiabatic paths. For a fluid as the working substance, the corresponding P - V diagram is given in the lecture notes.



[3mks]

(b) For the general reversible engine cycle shown below



we can express the area under a curve in the T - S plane as (I take T on the vertical axis and S on the horizontal axis)

$$Area = \int_{S_i}^{S_f} T dS = \int_i^f dQ = \text{Heat absorbed in going from } i \text{ to } f$$

We can split the engine cycle into an upper path starting at S_{min} and ending at S_{max} and the return (lower) path starting at S_{max} and finishing at S_{min} . On the upper path we have

$$A_{UP} = \int_{min}^{max} dQ = \Delta Q_{S_{min} \rightarrow S_{max}} \quad [4mks]$$

Since the entropy of the substance is increasing, it is absorbing heat so that the area computed just above is the total heat Q_1 supplied to the engine,

$$A_{UP} = Q_1$$

On the return path, the positive area A_{LOW} is also a heat flow given by

$$A_{LOW} = \left| \int_{max}^{min} dQ \right| = |-Q_2| \quad [4mks]$$

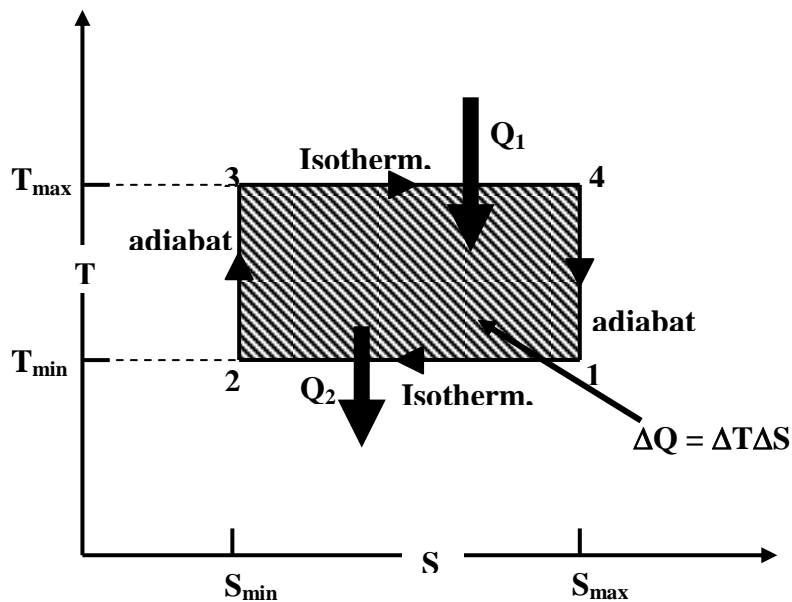
where Q_2 is the waste heat coming out of the engine,

$$A_{LOW} = Q_2$$

The engine efficiency is given then as

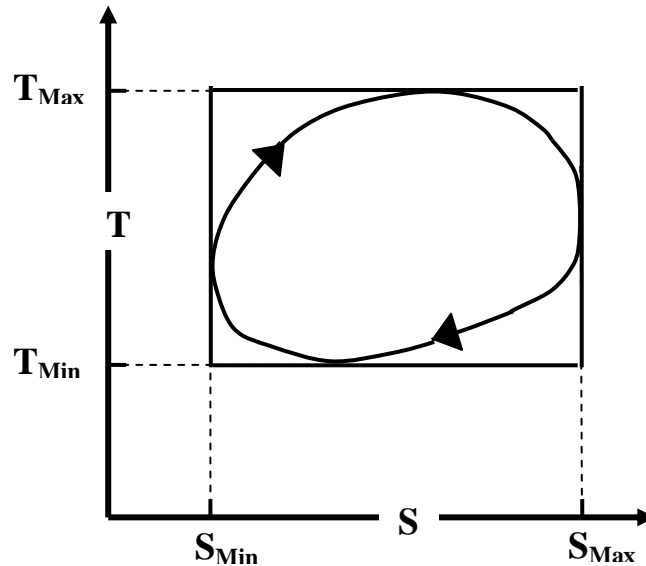
$$\eta_E = 1 - \frac{Q_2}{Q_1} = 1 - \frac{A_{LOW}}{A_{UP}} \quad [1mk]$$

(c) In the T - S plane, an isotherm is a horizontal straight line while an adiabat is a vertical straight line (constant entropy). Thus the Carnot cycle is just a rectangle.



[3mks]

(d) To compare the general engine shown in (b) with the Carnot engine shown in (c), enclose the general cycle in the smallest rectangle that will just hold it, i.e., a rectangle with horizontal sides at T_{\max} and T_{\min} and with vertical sides at S_{\max} and S_{\min}



By the geometry of this diagram it is obvious that the following inequalities hold:

$$A_{LOW}^{Engine} > A_{LOW}^{Carnot} \quad A_{UP}^{Carnot} > A_{UP}^{Engine} \quad [2mks]$$

It follows that

$$\frac{A_{LOW}^{Engine}}{A_{UP}^{Engine}} > \frac{A_{LOW}^{Carnot}}{A_{UP}^{Carnot}} \quad [2mks]$$

And hence that

$$1 - \frac{A_{LOW}^{Engine}}{A_{UP}^{Engine}} < 1 - \frac{A_{LOW}^{Carnot}}{A_{UP}^{Carnot}} \quad [2mks]$$

or

$$\eta_{General Engine} < \eta_{Carnot Engine}$$