Thermal & Kinetic Physics (PHY-214) Exam 2010 Solutions.

SECTION A

A1.
$$dU = TdS - PdV$$

dU is the incremental change in internal energy.

dS is the incremental change in entropy.

-PdV is the incremental work done <u>on the system</u> where P is the pressure and dV the incremental change in volume. [5]

A2.
$$S = k_B \ln \Omega$$
 [5]

A3.
$$\eta = 1 - \frac{T_2}{T_1}$$
 [5]

A4. $\Delta S \ge 0$ where $\Delta S = 0$ for a reversible process. [5]

A5.

$$\eta_R = \frac{Q_2}{W} = \frac{Q_2}{Q_1 - Q_2}$$
or
$$\frac{1}{Q_1/Q_2 - 1}$$

$$Q_1$$

$$Q_2$$

$$T_C$$

A6. $P_i V_i^{\gamma} = P_f V_f^{\gamma}$ where $\gamma = {^CP}/{c_V}$ is the ratio of the specific heat capacity at constant pressure, c_P to that at constant volume, c_V . [5]

A7.
$$C_P = \left(\frac{\partial H}{\partial T}\right)_P$$

H is the enthalpy, T the temperature and P the pressure of the system. [5]

A8.

$$U = \frac{5}{2} N k_B T$$
 [3]

$$\left\langle \frac{1}{2}mv^2 \right\rangle = \frac{3}{2}k_B T \tag{2}$$

A9. For an ideal gas PV = RT and therefore we can write

$$P = \frac{RT}{V}$$
 [1]

$$\left(\frac{\partial P}{\partial V}\right)_T = \frac{-RT}{V^2}$$
 [1]

$$\kappa = -V \left(\frac{\partial P}{\partial V} \right)_{V} = \frac{RT}{V} = P$$
 [3]

A10. dW = -PdV

$$\Delta W = -\int_{i}^{f} P dV = -P \int_{V_{i}}^{V_{f}} dV = P(V_{i} - V_{f}) = +P \frac{V}{2}$$
 [4]

The sign is positive and therefore represents work done ON the gas. [1]

SECTION B

<u>B1</u>

a)

i)
$$\Delta W = -\int\limits_{i}^{f} P dV$$

$$PV = RT \qquad \qquad P = \frac{RT}{V}$$

$$\Delta W = -\int_{i}^{f} P dV = -RT \int_{V_{i}}^{V_{f}} \frac{dV}{V} = -RT \left(\ln V_{f} - \ln V_{i} \right) = -RT \ln 2$$

As $\ln 2$ is greater than 0 it follows that $\Delta W < 0$ and this represents work done BY the gas.

[2]

ii) In an isochoric process there is no work done and therefore by the first law of thermodynamics we have

$$\Delta U = \Delta Q = \frac{3}{2}V_1 \left(\frac{P_1}{2} - P_1\right) = -\frac{3}{4}P_1V_1$$

This is a negative quantity indicating that heat has been released by the gas. [1]

iii)
$$\Delta U = \frac{5}{2} P_1 \left(\frac{V_1}{2} - V_1 \right) = -\frac{5}{4} P_1 V_1$$
 [1]

b)

i) From the adiabatic rule $P_i V_i^{\gamma} = P_f V_f^{\gamma}$ by using PV = RT and $P = \frac{RT}{V}$ we obtain the rule in terms of V and T

$$T_i V_i^{\gamma - 1} = T_f V_f^{\gamma - 1}$$

$$T_f = T_i \left(\frac{V_i}{V_f}\right)^{\gamma - 1} = 298.15 \times 2^{0.4} = 393.41K = 144.02 \, {}^{o}C_{-}$$
 [2]

$$U = \frac{5}{2}RT$$

$$\Delta U = \frac{5}{2} R \Delta T$$

$$\Delta U = \frac{5}{2}8.31J \ mol^{-1}K^{-1} \times 62.27 \ K = +1293.7J$$
 [2]

iii) There is no heat transfer during an adiabatic process andt the first law of thermodynamics then gives

$$\Delta W = \Delta U = +1293.7J$$

Being positive the work is done on the gas during this process. [2]

c)

i)
$$\eta_E = 1 - \frac{Q_2}{Q_1}$$
 [1]

ii) To find the engine efficiency we need to begin by finding Q_1 and Q_2 . To find Q_1 begin with the fact that $\mathbf{2} \to \mathbf{3}$ is an isochore and no work is done as $\mathrm{d}V = 0$. The heat can be calculated from the first law

There are two ways to proceed either is acceptable

$$Q_1 = \Delta Q_{32} = \Delta U_{32} = U_3 - U_2 = \frac{5}{2} nR(T_3 - T_2)$$

or alternatively

$$Q_1 = \Delta Q_{32} = \Delta U_{32} = \frac{5}{2} V_2 (P_3 - P_2)$$

We can see that Q_1 is positive as $T_3 > T_2$ $(P_3 > P_2)$ and it is therefore a flow of heat into the gas.

To find Q_2 we may use the fact that $\mathbf{4} \to \mathbf{1}$ is also an isochore and again no work is done. The heat can be calculated from the first law as before

$$\Delta Q_{14} = \Delta U_{14} = U_1 - U_4 = \frac{5}{2} nR(T_1 - T_4)$$

Or alternatively

$$\Delta Q_{14} = \Delta U_{14} = \frac{5}{2} V_1 (P_1 - P_4)$$

 ΔQ_{14} is negative as $T_1 < T_4$ ($P_1 < P_4$) and it is therefore a flow of heat from the gas as shown. However we need the modulus of ΔQ_{14} to represent Q_2 in the expression for

engine efficiency as this expression requires a positive value (the direction of heat flow is implicitly accounted for in the expression for efficiency and thus

$$Q_2 = -\Delta Q_{14} = \frac{5}{2} nR(T_4 - T_1)$$

Or

$$Q_2 = -\Delta Q_{14} = \frac{5}{2}V_1(P_4 - P_1)$$

Using our earlier expression for the efficiency

$$\eta_E = 1 - \frac{Q_2}{Q_1} = 1 - \frac{(T_4 - T_1)}{(T_3 - T_2)}$$

Or

$$\eta_E = 1 - \frac{Q_2}{Q_1} = 1 - \frac{V_1}{V_2} \frac{(P_4 - P_1)}{(P_3 - P_2)}$$

We need this in terms of the two volumes V_1 and V_2 and to achieve this use the fact that $\mathbf{1} \to \mathbf{2}$ is an adiabatic compression where work is done on the gas and the equation from \mathbf{b} \mathbf{i}) holds;

$$T_1 V_1^{\gamma - 1} = T_2 V_2^{\gamma - 1}$$

Or

$$P_1V_1^{\gamma} = P_2V_2^{\gamma}$$

Also $3 \rightarrow 4$ is an adiabatic expansion where work is done by the gas

$$T_4 V_4^{\gamma - 1} = T_3 V_3^{\gamma - 1}$$

Or

$$P_4V_4^{\gamma} = P_3V_3^{\gamma}$$

Subtracting the first of these adiabatic expressions from the second and noting that $V_1 = V_4$ and that $V_2 = V_3$

$$(T_4 - T_1)V_1^{\gamma - 1} = (T_3 - T_2)V_2^{\gamma - 1}$$

Or

$$(P_4 - P_1)V_1^{\gamma} = (P_3 - P_2)V_2^{\gamma}$$

therefore

$$\frac{(T_4 - T_1)}{(T_3 - T_2)} = \left(\frac{V_2}{V_1}\right)^{\gamma - 1}$$

Or

$$\frac{(P_4 - P_1)}{(P_3 - P_2)} = \left(\frac{V_2}{V_1}\right)^{\gamma}$$

and

$$\eta_E = 1 - \left(\frac{V_2}{V_1}\right)^{\gamma - 1}$$

$$\eta_E = 1 - \frac{1}{r^{\gamma - 1}}$$
QED [6]

iii) The compression ratio is

$$r = \frac{V_1}{V_2} = \frac{300cm^3}{75cm^3} = 4$$
 [2]

And therefore

1.
$$\eta_E = 1 - \frac{1}{r^{\gamma - 1}} = 1 - \frac{1}{4^{0.4}} = 0.425$$
 [2]

2.
$$P_2 = P_1 \left(\frac{V_1}{V_2}\right)^{\gamma} = P_1 r^{1.4} = 1 \quad atm \times 4^{1.4} = 6.96 \quad atm$$
 [2]

3.
$$Q_1 = \frac{5}{2}nR(T_3 - T_2) = \frac{5}{2}(P_3V_3 - P_2V_2) = \frac{5}{2}V_2(P_3 - P_2)$$
 [2]

$$Q_1 = \frac{5}{2}V_2(P_3 - P_2) = \frac{5}{2} \times 75 \times 10^{-6} \, m^3 (25 - 6.96) \times 1.01 \times 10^5 \, Pa = 341.62 \, J$$

4. Finally if $\eta_E = 0.425$ and $Q_1 = 341.62$ *J* we have

$$W = \eta_E Q_1 = 145 \ J \ per cycle$$

Therefore with 50 cycles per second

Power =
$$50 \times 145 \ Watts = 7.26 \ kW$$
 [2]

<u>B2.</u>

a)

$$\Delta S = S_f - S_i = \int_i^f \frac{\delta Q_R}{T}$$

The reversible heat transfer δQ_R is by convention positive when heat flows into the system whose entropy change is being considered. [2]

ii) The entropy change of the water as it cools is

$$\Delta S = \int_{i}^{f} \frac{\delta Q_R}{T}$$
 and $\delta Q_R = mC_P dT$

Therefore

$$\Delta S = mC_P \int_{303.15}^{280.15} \frac{dT}{T} = 3kg \times 4.2 \times 10^3 J kg^{-1} K^{-1} \times \ln \frac{280.15}{303.15}$$

$$\Delta S = -994 JK^{-1}$$
 [2]

- iii) This process occurs in three stages and the calculation must be split into three parts;
 - 1) The ice warming from -10 °C to 0 °C

$$\Delta S_1 = \int_{i}^{f} \frac{\delta Q_R}{T}$$
 and $\delta Q_R = mC_P dT$

$$\Delta S_1 = mC_P^{Ice} \int_{263.15}^{273.15} \frac{dT}{T} = 3kg \times 2.1 \times 10^3 \, Jkg^{-1} K^{-1} \times \ln \frac{273.15}{263.15}$$

$$\Delta S_1 = 236.0 \ JK^{-1}$$

2) The ice melting at constant temperature

$$\Delta S_2 = \frac{ml_{melt}}{T} = \frac{3kg \times 3.33 \times 10^5 J kg^{-1}}{273.15}$$

$$\Delta S_2 = 3657.33 \ JK^{-1}$$

3) The melted ice water warming up to ambient temperature

$$\Delta S_3 = mC_P \int_{273.15}^{280.15} \frac{dT}{T} = 3kg \times 4.2 \times 10^3 \, Jkg^{-1}K^{-1} \times \ln \frac{280.15}{273.15}$$

$$\Delta S_3 = 319.0 \ JK^{-1}$$

The net change in entropy is the sum of these three

$$\Delta S_{net} = 4212.33 \ JK^{-1}$$
 [5]

b)

i) We have the thermodynamic identity for a gas

$$dU = TdS - PdV$$

Also, for a monatomic ideal gas

$$U = \frac{3}{2}RT$$

Therefore

$$dU = \frac{3}{2}RdT$$

Re-arranging the expression for the thermodynamic identity

$$TdS = dU + PdV = PdV + \frac{3}{2}RdT$$

From the equation of state for an ideal gas

$$P = \frac{RT}{V}$$

$$TdS = RT\frac{dV}{V} + \frac{3}{2}RdT$$

Re-arranging

$$dS = R\frac{dV}{V} + \frac{3}{2}R\frac{dT}{T}$$

Integrating both sides at constant volume

$$\Delta S = R\frac{3}{2}\ln T + f(V)$$

Integrating both sides at constant temperature

$$\Delta S = R \ln T + g(T)$$

For both to be generally true in passing from initial temperature and volume, T_i , V_i to final temperature and volume, T_f , V_f

$$\Delta S = \frac{3}{2} R \ln \left[\frac{T_f}{T_i} \right] + R \ln \left[\frac{V_f}{V_i} \right]$$
 QED [5]

ii) In an isothermal process we have;

$$\Delta S = R \ln \frac{V_f}{V_i} = R \ln \frac{1}{3} = -8.31 \times 1.1 = -9.13 \ JK^{-1}$$
 [2]

iii) In an isochoric process we have

$$\Delta S = \frac{3}{2} R \ln \left[\frac{T_f}{T_i} \right] = 1.5 \times 8.31 \times \ln 2 = +8.63 \ JK^{-1}$$
 [2]

c)
i) The system will reach equilibrium by reaching the temperature of the reservoir and in this process

$$\Delta S_{system} = \int_{T_i}^{T_R} \frac{\delta Q}{T} = mC_P \int_{T_i}^{T_R} \frac{dT}{T} = mC_P \ln \left[\frac{T_R}{T_i} \right]$$

And the reservoir remains at a constant temperature giving up heat ΔQ to the system and therefore

$$\Delta S_{reservoir} = \int_{i}^{f} \frac{dQ_R}{T} = \frac{1}{T_R} \int_{i}^{f} dQ_R = \frac{\Delta Q_R}{T_R} = -\frac{mC_P}{T_R} (T_R - T_i)$$

$$\Delta S_{Universe} = \Delta S_{System} + \Delta S_{reservoir}$$

$$\Delta S_{Universe} = mC_P \ln \left(\frac{T_R}{T_i}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \left(1 - \frac{T_i}{T_R}\right) = -mC_P \ln \left(\frac{T_i}{T_R}\right) - mC_P \ln \left(\frac{T_i}{T_R}\right) = -mC_P \ln \left($$

which can be re-written as

$$\Delta S_{Universe} = -mC_P \ln \left[1 - \left(1 - \frac{T_i}{T_R} \right) \right] - mC_P \left(1 - \frac{T_i}{T_R} \right)$$

This is in the form

$$\Delta S_{Universe} = -mC_P \ln(1-X) - mC_P X$$

where

$$X = \left(1 - \frac{T_i}{T_R}\right) \qquad \qquad \mathbf{QED} \quad [5]$$

ii) For the system warming, $\frac{T_i}{T_R} < 1$ and X is positive.

Using McClaurens series expansion, $-\ln(1-x) = x + \frac{x^2}{2} + \frac{x^3}{3} + \dots$

$$\Delta S_{Universe} = mC_P \left[X + \frac{X^2}{2} + \frac{X^3}{3} + \dots - X \right] = mC_P \left[\frac{X^2}{2} + \frac{X^3}{3} + \dots \right] > 0$$

 $\Delta S_{Universe}$ has increased. **QED** [2]

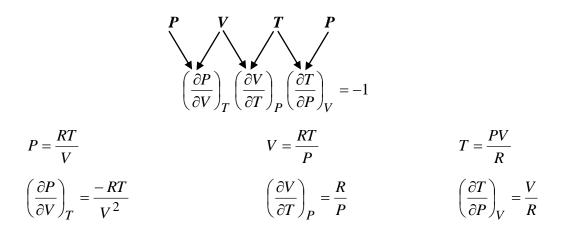
<u>B3.</u>

a)

i) Starting with the ideal gas equation of state

$$PV = RT$$

The three variables are P,V,T and we form the cyclic relation



$$\left(\frac{\partial P}{\partial V}\right)_{T}\left(\frac{\partial V}{\partial T}\right)_{P}\left(\frac{\partial T}{\partial P}\right)_{V} = \frac{-RT}{V^{2}}\frac{R}{P}\frac{V}{R} = \frac{-PV}{V^{2}}\frac{R}{P}\frac{V}{R} = -1 \qquad \mathbf{QED} \quad [2]$$

ii) The product $\kappa\beta$ may be written using the given definitions of the two quantities as

$$\kappa \beta = -\left(\frac{\partial P}{\partial V}\right)_T \left(\frac{\partial V}{\partial T}\right)_P$$

We have already from part i) using the cyclical relation

$$\left(\frac{\partial P}{\partial V}\right)_T \left(\frac{\partial V}{\partial T}\right)_P \left(\frac{\partial T}{\partial P}\right)_V = -1$$

Therefore using the reciprocal relation

$$\kappa\beta = -\left(\frac{\partial P}{\partial V}\right)_T \left(\frac{\partial V}{\partial T}\right)_P = \frac{1}{\left(\frac{\partial T}{\partial P}\right)_V} = \left(\frac{\partial P}{\partial T}\right)_V$$

The required differential relationship.

iii) The pressure is changed isothermally and therefore use $\kappa = -V \left(\frac{\partial P}{\partial V} \right)_T$ to find the change in temperature by considering small changes

[2]

$$\kappa = -V \left(\frac{\partial P}{\partial V} \right)_T = -\frac{V}{\Delta V} \Delta P$$

We also have from part i)
$$\left(\frac{\partial P}{\partial V}\right)_T = \frac{-RT}{V^2}$$
 \Rightarrow $\left(\frac{\partial P}{\partial V}\right)_T = \frac{-PV}{V^2}$

Therefore

$$\kappa = -V \left(\frac{\partial P}{\partial V} \right)_T = P = -\frac{V}{\Delta V} \Delta P$$

And

$$\frac{V}{\Delta V} = -\frac{P}{\Delta P} = -0.02$$

The volume also changes decreasing by -2%.

[2]

b)

i) Given the definition of the Gibbs free energy, G = U - TS + PV

To find the natural variables we write the infinitesimal of G

$$dG = dU - TdS - SdT + PdV + VdP$$

Now use the thermodynamic identity, dU - TdS + PdV = 0 to tidy this up

$$dG = -SdT + VdP$$

The natural variables of G are then T and P

$$G = G(T, P)$$
 [4]

ii) Using the natural variables we may write the infinitesimal of G as

$$dG = \left(\frac{\partial G}{\partial T}\right)_{P} dT + \left(\frac{\partial G}{\partial P}\right)_{T} dP$$

And compare this with the expression of dG from i)

$$\left(\frac{\partial G}{\partial T}\right)_P = -S$$
 and $\left(\frac{\partial G}{\partial P}\right)_T = V$ [3]

iii) From i) and ii) and using the fact that dG is a perfect differential and that T and P are the natural variables of G gives

$$\frac{\partial}{\partial P} \left(\frac{\partial G}{\partial T} \right)_{P} = \frac{\partial}{\partial T} \left(\frac{\partial G}{\partial P} \right)_{T}$$

And therefore

$$-\frac{\partial S}{\partial P} = \frac{\partial V}{\partial T}$$

Which is the required Maxwell relation.

[3]

c)

i) In part a the thermal expansion coefficient was defined as

$$\beta = \frac{1}{V} \left(\frac{\partial V}{\partial T} \right)_{P}$$

We are given the equation of state for one mole of a van der Waals gas,

$$\left(P + \frac{a}{V^2}\right)(V - b) = RT$$

We need to find $\left(\frac{\partial V}{\partial T}\right)_P$

Begin by finding $\left(\frac{\partial T}{\partial V}\right)_P$

$$\left(\frac{\partial T}{\partial V}\right)_{P} = \frac{1}{R} \left(\frac{-2a}{V^{3}} (V - b) + P + \frac{a}{V^{2}}\right)$$

$$\left(\frac{\partial T}{\partial V}\right)_{P} = \frac{1}{R} \left(\frac{-2a}{V^{2}} + \frac{2ab}{V^{3}} + P + \frac{a}{V^{2}}\right)$$

We need to use the reciprocal relation and therefore to invert this equation and start by getting the RHS over a common denominator

$$\left(\frac{\partial T}{\partial V}\right)_{P} = \frac{1}{R} \left(\frac{-2aV}{V^{3}} + \frac{2ab}{V^{3}} + \frac{PV^{3}}{V^{3}} + \frac{aV}{V^{3}}\right) = \frac{PV^{3} - aV + 2ab}{RV^{3}}$$

$$\left(\frac{\partial V}{\partial T}\right)_{P} = \frac{RV^{3}}{PV^{3} - aV + 2ab}$$

And therefore following the definition given we have

$$\beta = \frac{1}{V} \left(\frac{\partial V}{\partial T} \right)_P = \frac{RV^2}{PV^3 - aV + 2ab}$$
 [4]

ii) Given that the internal energy of a van der Waals gas is;

$$U_{VdW} = \frac{3}{2}RT - \frac{a}{V}$$

We can obtain dU and use this in the first law

$$dU = \frac{3}{2}RdT + \frac{a}{V^2}dV$$

The first law is

$$dU = \delta Q - PdV$$

And for an adiabatic process

$$dU = -PdV$$

therefore

$$\frac{3}{2}RdT + \frac{a}{V^2}dV = -PdV$$

From the van der Waals equation of state, $\left(P + \frac{a}{V^2}\right)(V - b) = RT$

$$P = \frac{RT}{V - b} - \frac{a}{V^2}$$

Using this expression for P

$$\frac{3}{2}RdT + \frac{a}{V^2}dV = \frac{-RT}{V-b}dV + \frac{a}{V^2}dV$$

$$\frac{3}{2}RdT = \frac{-RT}{V - b}dV$$

$$\frac{3}{2}\frac{dT}{T} = \frac{-dV}{V - b}$$

Integrating both sides

$$\frac{3}{2}\ln T = \ln T^{\frac{3}{2}} = -\ln(V - b) + const$$

$$\ln T^{\frac{3}{2}} + \ln(V - b) = const$$

$$T^{\frac{3}{2}}(V - b) = const$$

$$QED$$
 [5]

<u>B4</u>

a)

$$v_{rms} = \sqrt{\frac{3k_BT}{m}}$$
 [2]

ii) Using the equation from part i) we find the room mean square velocity of N_2

$$v_{rms} = \sqrt{\frac{3k_BT}{m}} = \sqrt{\frac{3 \times 1.38 \times 10^{-23} JK^{-1} \times 300K}{28 \times 1.66 \times 10^{-27}}}$$
$$v_{rms}(N_2) = \sqrt{2.67 \times 10^5} = 517 \text{ ms}^{-1}$$
 [2]

Noting that the velocity depends on $\sqrt{m^{-1}}$ and that the mass of O_2 is a factor of $\frac{32}{28} = 1.1428$ larger, the velocity will be a factor 1.069 smaller

$$v_{rms}(O_2) == \frac{517 \text{ ms}^{-1}}{1.069} = 483.6 \text{ ms}^{-1}$$
 [2]

b)

i) The Maxwell speed distribution is $P(v)dv = 4\pi v^2 f(v)dv$

$$P(v) = \left(\frac{m}{2\pi k_B T}\right)^{3/2} 4\pi v^2 \exp\left[-\frac{mv^2}{2k_B T}\right]$$
 [2]

ii) To find the mean of a quantity requires integration of the product of that quantity with the probability distribution over all possible values;

$$\overline{v} = \int_{0}^{\infty} v P(v) dv$$
 [2]

iii) Making the suggested substitution

$$v^2 \to x$$
, $dv = \frac{dx}{2v}$

and writing $a = \frac{m}{2k_BT}$ the integral of part ii) becomes

$$\overline{v} = 4\pi \left[\frac{a}{\pi}\right]^{3/2} \frac{1}{2} \int_{0}^{\infty} x \exp(-ax) dx = 2\pi \left[\frac{a}{\pi}\right]^{3/2} \int_{0}^{\infty} x \exp(-ax) dx$$

We can find the integral, $\int_{0}^{\infty} x \exp(-ax) dx$, by integrating by parts; $\int u dv = uv - \int v du$

as follows;

$$u = x$$
 $dv = e^{-ax}$

$$du = dx v = -\frac{1}{a}e^{-ax}$$

$$\int u dv = \int x e^{-ax} = -\frac{x}{a} e^{-ax} - \int -\frac{1}{a} e^{-ax} dx$$

$$\int_{0}^{\infty} xe^{-ax} dx = \left[-\frac{x}{a}e^{-ax} - \frac{1}{a^{2}}e^{-ax} \right]_{0}^{\infty} = \frac{1}{a^{2}}$$

$$\bar{v} = 2\pi \left[\frac{a}{\pi}\right]^{3/2} \frac{1}{a^2} = \sqrt{\frac{4}{\pi a}} = \sqrt{\frac{8k_B T}{\pi m}}$$
 [6]

c)

i) We are given the expression for the flux, $\Phi = \frac{1}{4} n \bar{v}$ where $n = \frac{N}{V}$.

We may get the flux in terms of temperature and pressure by using the ideal gas equation of state,

$$PV = Nk_BT$$

Therefore

$$n = \frac{N}{V} = \frac{P}{k_B T}$$

And the temperature will enter also through the mean velocity as found in part i),

$$\overline{v} = \sqrt{\frac{8k_BT}{\pi m}}$$

Therefore the flux is

$$\Phi(P,T) = \frac{1}{4} \frac{P}{k_B T} \sqrt{\frac{8k_B T}{\pi m}} = \frac{P}{\sqrt{2\pi m k_B T}}$$
 [3]

ii) In the effusion process described every gas molecule incident upon the hole will escape. Thus the number that escape per unit time of molecular species i is given by,

$$N_i^e = \Phi_i A = \frac{1}{4} n_i \sqrt{\frac{8k_B T}{\pi m_i}}$$

Therefore

$$\frac{N_1^e}{N_2^e} = \frac{n_1}{n_2} \sqrt{\frac{m_2}{m_1}}$$
 QED [3]

iii) From part ii) we find the ratio of Helium isotopes as

$$\frac{N_{3}^{e}_{He}}{N_{4He}^{e}} = \frac{n_{3}_{He}}{n_{4}_{He}} \sqrt{\frac{m_{4}_{He}}{m_{3}_{He}}} = \frac{10}{90} \sqrt{\frac{4}{3}} = 0.128$$

This tells us that $N_{3He}^e = 0.128 N_{4He}^e$ and therefore that

 $N_{3He}^{e} + N_{4He}^{e} = 1.128N_{4He}^{e}$ so the percentage of ³He in the escaped mixture is

$$\frac{N_{3}^{e}_{He}}{N_{3}^{e}_{He} + N_{4}^{e}_{He}} \times 100 = \frac{0.128}{1.128} \times 100 = 11.4\%$$
 [3]